Real Time Systems and Control Applications



Contents PID Controller

Stability Range of K?

Consider an open loop transfer function G(s)= K/s(s+1)(s + 2). What K values make the closed loop system stable?



Need to get K so that the root locus intercept imaginary axis.

Consider the characteristic equation: 1+K/s(s+1)(s + 2)=0

So $s^3 + 3s^2 + 2s + K = 0$

Assuming $s = \beta j$, solve K=6 when root locus intercepts the imagery axis, so the stability range of K is (0, 6).

Proportional Control

Proportional control action is obtained when we multiply the error signal by a constant gain, i.e. our control input to the plant has a value proportional to the value of the error:

$$K_p e(t) = K_p[u(t) - y(t)]$$



Example



- Closed loop transfer function is: $T(s) = \frac{G_p(s)}{1+G_p(s)} = \frac{1}{s+2}$
- Time constant: $\frac{1}{2}$ second
- The steady state output is: $y_{ss}(t) = \lim_{s \to 0} s T(s) u(s) = \frac{1}{2}$
- The steady state error is : $\frac{1}{2}$

Adding Proportional Control

• Closed loop Transfer Function is

$$T(s) = \frac{K_p G_p}{1 + K_p G_p}$$

For
$$K_p = 9$$
 and $G_p = \frac{1}{s+1} \Rightarrow T(s) = \frac{9}{s+10}$, then the steady state output is:

$$Y(s) = \lim_{s \to 0} (s T(s) u(s)) = \lim_{s \to 0} \frac{9}{s + 10} = 0.9$$

The steady state error has been changed to 0.1, and au=0.1

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• The closed loop Transfer Function is $T(s) = \frac{G_c G_p}{1 + G_c G_p} = \frac{K_I}{s^2 + s + K_I}$

The steady state output is

$$Y(s) = \lim_{s \to 0} (s T(s) u(s)) = \lim_{s \to 0} \frac{K_I}{s^2 + s + K_I} = 1$$

So the steady state error is 0.

Proportional-Integral (PI) Control



- Closed loop transfer function becomes: $T(s) = \frac{K_I + sKp}{s^2 + (1 + Kp)s + K_I}$
- Combination of proportional control and integral control
 - Proportional control has impact on speed of response
 - Integral control is used to force steady state error to zero



Closed loop transfer function becomes:

$$T(s) = \frac{G_c G_p}{1 + G_c G_p} = \frac{K_D s}{(1 + K_D)s + 1}$$

• Derivative action introduces an open loop zero, as K_D increases the system may not be stable.



• Closed loop transfer function becomes:

$$T(s) = \frac{G_c G_p}{1 + G_c G_p} = \frac{s K_D P \omega_n^2}{s^2 + 2\left(\xi + \frac{K_D}{2} P \omega_n\right) \omega_n s + \omega_n^2}$$

• Derivative action introduces or changes damping effect, $\xi' = \xi + \frac{K_D}{2} P \omega_n$.

PID Control

• PID is an abbreviation for Proportional/Integral/Derivative Control.



Effects of PID Control

- Proportional Gain: steady state system error is reduced by increasing Kp.
- Integral Gain: we can use integral action to reduce steady state error to zero. The trade-off here is again stability and dynamic performance.
- Derivative Gain: Settling/peak time and damping in our system is directly affected by our derivative gain.

PID Controller

• In time domain:



- For proportional part, $u(k) = K_P e(k)$ For integral part, $\frac{du(t)}{dt} = K_I e(t) \Rightarrow \frac{u(k) u(k-1)}{T} = K_I e(t)$

$$u(k) = u(k-1) + K_I Te(k) = u(k-2) + K_I T(e(k-1) + e(k))$$

• As initial value u(0) = 0, the contribution from integral component can be written as: 1

$$u(k) = K_I T \sum_{i=1}^{k} e(i)$$

PID Controller (continued)

• The derivative part approximates to K_D

$$u(k) = \frac{\pi_D}{T} \left[e(k) - e(k-1) \right]$$

• Therefore, the PID controller approximates to:

$$u(k) = K_P e(k) + K_I T \sum_{i=1}^{\kappa} e(i) + \frac{K_D}{T} [e(k) - e(k-1)]$$

This equation is used to implement proportional, integral or derivative controllers in software!

Question

• Show that PI controller cannot stabilize $G_p(s) = \frac{1}{(s-1)^2}$