# Real Time Systems and Control Applications



Contents PID Controller

#### Stability Range of K?

• Consider an open loop transfer function G(s)= K/s(s+1)(s + 2). What K values make the closed loop system stable?



Need to get K so that the root locus intercept imaginary axis.

Consider the characteristic equation:  $1 + K/s(s+1)(s + 2)=0$ 

So  $s^3 + 3s^2 + 2s + K = 0$ 

Assuming  $s = \beta j$ , solve K=6 when root locus intercepts the imagery axis, so the stability range of K is (0, 6).

#### Proportional Control

Proportional control action is obtained when we multiply the error signal by a constant gain, i.e. our control input to the plant has a value proportional to the value of the error:

$$
K_p e(t) = K_p[u(t) - y(t)]
$$



### Example



- Closed loop transfer function is:  $T(s) =$  $G_p(S)$  $1+G_p(s)$ = 1 s+2
- Time constant:  $\frac{1}{2}$ 2 second
- The steady state output is:  $y_{SS}(t) = \lim_{s \to 0}$  $S\rightarrow 0$  $S T(S) u(S) =$  $\overline{\mathsf{I}}$ 2
- The steady state error is :  $\frac{1}{2}$ 2

 $Y(s) = \lim_{s \to 0}$  $S\rightarrow 0$  $(s T(s) u(s)) = \lim_{s \to 0}$  $S\rightarrow 0$ 9  $s + 10$  $= 0.9$ The steady state error has been changed to 0.1, and  $\tau = 0.1$ 

For  $K_p = 9$  and  $G_p = \frac{1}{s+1}$ s+1  $\rightarrow T(s) = \frac{9}{s+10}$ , then the steady state output is:

\n- Closed loop Transfer Function is 
$$
T(s) = \frac{K_p G_p}{1 + K_p G_p}
$$
\n

# Adding Proportional Control





• The closed loop Transfer Function is  $T(s) =$  $G_{\cal C}G_{\cal P}$  $1 + G_c G_p$ =  $K_I$  $s^2 + s + K_I$ 

The steady state output is

$$
Y(s) = \lim_{s \to 0} (s T(s) u(s)) = \lim_{s \to 0} \frac{K_I}{s^2 + s + K_I} = 1
$$

#### So the steady state error is 0.

### Proportional-Integral (PI) Control



- Closed loop transfer function becomes:  $T(s) =$  $K_I + s$  $s^2 + (1 + Kp)s + K_I$
- Combination of proportional control and integral control
	- Proportional control has impact on speed of response
	- Integral control is used to force steady state error to zero



• Closed loop transfer function becomes:

$$
T(s) = \frac{G_c G_p}{1 + G_c G_p} = \frac{K_D s}{(1 + K_D)s + 1}
$$

• Derivative action introduces an open loop zero, as  $K<sub>D</sub>$  increases the system may not be stable.



• Closed loop transfer function becomes:

$$
T(s) = \frac{G_c G_p}{1 + G_c G_p} = \frac{s K_D P \omega_n^2}{s^2 + 2\left(\xi + \frac{K_D}{2} P \omega_n\right) \omega_n s + \omega_n^2}
$$

• Derivative action introduces or changes damping effect,  $\xi' = \xi + \frac{K_D}{2}$  $\frac{d^2D}{2}P\omega_n$ .

#### PID Control

• PID is an abbreviation for Proportional/Integral/Derivative Control.



#### Effects of PID Control

- Proportional Gain: steady state system error is reduced by increasing Kp.
- Integral Gain: we can use integral action to reduce steady state error to zero. The trade-off here is again stability and dynamic performance.
- Derivative Gain: Settling/peak time and damping in our system is directly affected by our derivative gain.

## PID Controller

• In time domain:



- 0<br>. • For proportional part,  $u(k) = K_P e(k)$
- For integral part,  $\frac{du(t)}{dt}$  $\frac{u(t)}{dt} = K_I e(t) \implies \frac{u(k)-u(k-1)}{T} = K_I e(t)$

 $u(t) = K_P e(t) + K_I$ 

$$
u(k) = u(k-1) + K_I Te(k) = u(k-2) + K_I T(e(k-1) + e(k))
$$

t<br>-

 $e(\eta)d\eta+K_D$ 

 $de(t)$ 

 $\boldsymbol{d}$ 

• As initial value  $u(0) = 0$ , the contribution from integral component can be written as:

$$
u(k) = K_I T \sum_{i=1}^k e(i)
$$

#### PID Controller (continued)

- The derivative part approximates to  $u(k) =$  $\frac{K_D}{\sqrt{2}}$  $\frac{1}{T} [e(k) - e(k-1)]$
- Therefore, the PID controller approximates to:

$$
u(k) = K_P e(k) + K_I T \sum_{i=1}^{k} e(i) + \frac{K_D}{T} [e(k) - e(k-1)]
$$

This equation is used to implement proportional, integral or derivative controllers in software!

#### Question

• Show that PI controller cannot stabilize  $G_p(s) = \frac{1}{(s-1)^2}$  $(s-1)^2$